Creep behavior of V-notched components

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ABSTRACT. Geometrical discontinuities such as notches play a significant role in structural integrity of the components, especially when the component is subjected to very severe conditions, such as the high temperature fatigue or creep. In this paper, a generalized form of the existing notch tip creep stress-strain analysis method developed by Nuñez and Glinka, is developed and extended to a wide variety of blunt V-notches. Assuming the generalized Lazzarin-Tovo solution that allows a unified approach to the evaluation of linear elastic stress fields in the vicinity of both cracks and notches is the key in getting the extension to blunt V-notches. Numerous cases have been analysed and the stress fields obtained according to the proposed method were compared with proper finite element data, showing a very good agreement.

KEYWORDS. Creep; V-notches; Stress fields; Stress evaluation; Strain energy density.

INTRODUCTION

Due to the complexities of the geometry and loading conditions in modern structural components, it is essential to be able to predict the behavior of components including geometrical discontinuities that generate localized high stress concentration zones [1,7]. They become even more important when, in operating conditions, the component is subjected to very demanding conditions such as high temperature fatigue or creep loading. Various methods have been proposed by researchers to evaluate the behavior of structural components under various loading conditions [8-12]. The structural components show a nonlinear stress-strain response such as creep (visco-plasticity) under applied load in a high temperature environment. In presence of geometric discontinuities such as notches, localized-creep takes place in a small region near the notch root. On the other hand, non-localized (or gross) creep condition refers to situations in which the far stress field also experiences some creep which may lead to more intense creeping in the vicinity of the notch tip.
To the best of the authors’ knowledge only a few solutions related to the localized time-dependent creep-plasticity problems are available in literature. Nuñez and Glinka [13] proposed a solution for non-localized creep strains/stresses at the notch root, based on the linear-elastic behavior of the material, the constitutive law and the material creep model. The formulation was derived by using the total strain energy density rule proposed by Neuber [14]. Considering the U-notched specimens ($2\alpha=0$ and $\rho \neq 0$) very good results were obtained using this method. The main aim of the current paper is to extend the method proposed by Nuñez and Glinka to blunt V-notches. For this aim, the Creager and Paris [15] equations were substituted with the Lazzarin and Tovo [16] equations. Finally an approach for fast evaluation of the stresses/strains at notches under non-localized creeping condition is proposed which doesn’t require any complex and time-consuming FE non-linear analyses. Output of the proposed approach can be used as input for creep life prediction models based on local approaches.

**EVALUATION OF STRESSES AND STRAINS UNDER NON-LOCALIZED CREEPING CONDITION FOR BLUNT V-NOTCHES**

Nuñez and Glinka [13] presented a method for the estimation of stress and strain at U-notch tip, subjected to non-localized creep. The method was based on the Neuber [14] concept extended to time dependent plane stress problems and on the introduction of $K_{\Omega}$ parameter introduced by Mofakhar et al. [17]. It can be assumed in fact that the total strain energy density changes occurring in the far field produce magnified effects at the notch tip. For this reason, the total strain energy density concentration factor is introduced in order to magnify the energy at the notch tip. The introduction of this parameter and of the far field stress and strain contribution in the Neuber’s time dependent formulation is the main difference within the non-localized and localized creep formulation that, instead, can be easily derived directly by extending the Neuber’s rule. Details about the original formulation can be found in the original works Nuñez and Glinka [13] and in Gallo et al. [18]. The key to extend the Nuñez-Glinka method to blunt V-notches is the assumption of the Lazzarin and Tovo [16] equations to describe the early elastic state of the system.

![Figure 1](image)

**Figure 1**: (a) Coordinate system and symbols used for the stress field components in Lazzarin-Tovo equations; (b) coordinate system and symbols used for the elastic stress field redistribution for blunt V-notches.

The Lazzarin-Tovo equations, in the presence of a traction loading, along the bisector (x axis), can be expressed as follows, as a function of the maximum stress (see Fig. 1):

$$
\begin{align*}
\{\sigma_\theta\} &= \sigma_{\text{max}} \left(\frac{r}{r_0}\right)^{\lambda-1} \left[1 + \Lambda_1 + \chi_1(1-\Lambda_1) + \left(\frac{r}{r_0}\right)^{\lambda_1} \left[3 - \Lambda_1 - \chi_1(1-\Lambda_1)\right]\right] \\
\{\sigma_r\} &= \frac{\sigma_{\text{max}}}{4} \left(\frac{r}{r_0}\right)^{\lambda-1} \left[3 - \Lambda_1 - \chi_1(1-\Lambda_1) - \left(\frac{r}{r_0}\right)^{\lambda_1} \left[3 - \Lambda_1 - \chi_1(1-\Lambda_1)\right]\right]
\end{align*}
$$

(1)
where $\sigma_{\text{max}}$ can be expressed as a function of stress concentration factor $K_t$ (evaluated through linear elastic finite element analysis) and the applied load $\sigma_{\text{nom}}$:

$$\sigma_{\text{max}} = K_t \sigma_{\text{nom}}$$  \hspace{1cm} (2)

Employing the more general conformal mapping of Neuber [19] that permit a unified analysis of sharp and blunt notches, the notch radius, $\rho$, and the origin of the coordinate system, $r_0$, are related by the following equation on the basis of trigonometric considerations:

$$\rho = \frac{q \cdot r_0}{q - 1}$$  \hspace{1cm} (3)

where $q = \frac{2\pi - 2\alpha}{\pi}$.

The main steps to extend the method to blunt V-Notches can be summarised as follows:

- Assumption of Lazzarin-Tovo equations to describe the stress distribution ahead the notch tip instead of Creager-Paris equations;
- Calculation of the origin of the coordinate system, $r_0$, as a function of the opening angle and notch radius, as described by Eq. (3);
- Re-definition of the plastic zone correction factor $C_p$ that is a function of plastic zone size $r_p$ and plastic zone increment $\Delta r_p$;

The definition of the parameters $C_p$, $r_p$, $\Delta r_p$ is very similar to that clearly reported by Glinka [20], except for the assumption of different elastic stress distribution equations. Definition of these variables is briefly reported hereafter.

Referring to Fig. 2, considering the Von Mises [21] yield criterion in polar coordinate:

$$\sigma_y = \sqrt{\sigma_r^2 - \sigma_\theta^2 + \sigma_\phi^2}$$  \hspace{1cm} (4)

and introducing Eqs. (1) into Eq. (4), a first approximation of $r_p$ that can be solved numerically is obtained. Once $r_p$ is known, the force $F_1$ can be evaluated as follows:

$$F_1 = \int_{r_0}^{r_p} \sigma_d r - \sigma_\theta \left( r_p - r_0 \right) =$$

$$\frac{K_t \sigma_{\text{nom}}}{4} \left[ \left( r_0 - r_p \right) \left( \frac{r_p}{r_0} \right)^{n-1} \left[ (\lambda + 1) + \lambda (1 - \lambda) \left( 1 - \frac{r_p}{r_0} \right)^{n-1} \right] \right] + \left( 3 - \lambda \right) \left( \frac{r_p}{r_0} \right)^{n-1}$$

$$- \frac{\left( \lambda + 1 \right) + \lambda (1 - \lambda)}{\lambda} \left[ \left( r_0 - r_p \right) \left( \frac{r_p}{r_0} \right)^{n-1} \right] \frac{1}{\mu}$$

The stress $\sigma_y (r_p)$ is considered to be constant inside the plastic zone, which means elastic-perfectly plastic behavior is assumed. The lower integration limit is $r_0$, which depends on the opening angle and notch tip radius. Due to the plastic yielding at the notch tip, the force $F_1$ cannot be carried through by the material in the plastic zone $r_p$. But in order to satisfy the equilibrium conditions of the notched body, the force $F_1$ has to be carried through by the material beyond the plastic zone $r_p$. As a result, stress redistribution occurs, increasing the plastic zone $r_p$ by an increment $\Delta r_p$. If the plastic zone is small in comparison to the surrounding elastic stress field, the redistribution is not significant, and it can be interpreted as a shift of the elastic field over the distance $\Delta r_p$ away from the notch tip. Therefore the force $F_1$ is mainly carried through the material over the distance $\Delta r_p$ and therefore the force $F_2$ (represented by the area depicted in the Fig.
1-b) must be equal to $F_1$. For this reasons, $F_1 = F_2 = \sigma_0 \Delta \lambda_0$, and the plastic zone increment can be expressed as the ratio between $F_1$ and $\sigma_\theta$ evaluated (through Lazzarin-Tovo equations [16]) at a distance equal to the previously calculated $r_p$:

$$\Delta r_p = \frac{F_1}{\sigma_\theta(r_p)}$$

(6)

Substituting in Eq. (6) the formula given by Eq. (5) for $F_1$ and the explicit form of $\sigma_\theta$, the expression for the evaluation of $\Delta r_p$ is obtained:

$$\Delta r_p = \left\{ \left( \frac{r_p}{r_0} \right)^{k-1} \left[ (r_0 - r_p) \left( \frac{r_p}{r_0} \right)^{k-1} \right] \left[ (\lambda_1 + 1) + \chi \left( 1 - \lambda_1 \right) \left[ 1 - \left( \frac{r_p}{r_0} \right)^{k-1} \right] + (3 - \lambda_1) \left( \frac{r_p}{r_0} \right)^{k-1} \right] - $$

$$- \left[ (\lambda_1 + 1) + \chi \left( 1 - \lambda_1 \right) \left[ r_0 - r_p \left( \frac{r_p}{r_0} \right)^{k-1} \right] \left[ \chi \left( 1 - \lambda_1 \right) \left( 3 - \lambda_1 \right) \left[ r_0 - r_p \left( \frac{r_p}{r_0} \right)^{k-1} \right] \right] \right] \right\} \frac{r_p}{\lambda_1} \left[ 1 - \left( \frac{r_p}{r_0} \right)^{k-1} \right] + (3 - \lambda_1) \left( \frac{r_p}{r_0} \right)^{k-1} \right\}$$

(7)

The last step consists in the definition of the plastic zone correction factor $C_p$, which is according to Glinka [20] but introducing the Lazzarin-Tovo equations:

$$C_p = 1 + \frac{\Delta r_p}{r_p} = 1 + \frac{\left( \frac{r_p}{r_0} \right)^{k-1} \left[ (r_0 - r_p) \left( \frac{r_p}{r_0} \right)^{k-1} \right] \left[ (\lambda_1 + 1) + \chi \left( 1 - \lambda_1 \right) \left[ 1 - \left( \frac{r_p}{r_0} \right)^{k-1} \right] + (3 - \lambda_1) \left( \frac{r_p}{r_0} \right)^{k-1} \right] - $$

$$- \left[ (\lambda_1 + 1) + \chi \left( 1 - \lambda_1 \right) \left[ r_0 - r_p \left( \frac{r_p}{r_0} \right)^{k-1} \right] \left[ \chi \left( 1 - \lambda_1 \right) \left( 3 - \lambda_1 \right) \left[ r_0 - r_p \left( \frac{r_p}{r_0} \right)^{k-1} \right] \right] \right] \right\} \frac{r_p}{\lambda_1} \left[ 1 - \left( \frac{r_p}{r_0} \right)^{k-1} \right] + (3 - \lambda_1) \left( \frac{r_p}{r_0} \right)^{k-1} \right\}$$

(8)

At this point, the general stepwise procedure to be followed to generate a solution is identical to that proposed by Nuñez and Glinka [13]:

1. Determine the notch tip stress, $\sigma_{22}$, and strain, $\varepsilon_{22}$, using the linear-elastic analysis.
2. Determine the elastic-plastic stress, $\sigma_{22}^0$, and strain, $\varepsilon_{22}^0$, using the Neuber [14] rule (or other methods e.g. ESED by Molski and Glinka [22], finite element analysis).
3. Begin the creep analysis by calculating the increment of creep strain, $\Delta \varepsilon_{22}^c$, for a given time increment $\Delta t$. The selected creep hardening rule has to be followed.

$$\Delta \varepsilon_{22}^c = \Delta t \varepsilon_{22}^c (\sigma; t)$$

(9)

4. Determine the decrement of stress, $\Delta \sigma_{22}^c$, from Eq. (10), due to the previously determined increment of creep strain, $\Delta \varepsilon_{22}^c$:
5. For a given time increment $\Delta t_n$, determine from Eq. (11) the increment of the total strain at the notch tip, $\Delta \varepsilon_{22}^n$:

$$\Delta \varepsilon_{22}^n = \Delta \varepsilon_{22}^n - \frac{\Delta \sigma_{22}^n}{E}$$

(11)

6. Repeat steps from 3 to 5 over the required time period.

RESULTS

The proposed new method has been applied to a hypothetical plate weakened by lateral symmetric V-notches, under Mode I loading; see Fig 1b. The notch tip radius $\rho$ and the opening angle $2\alpha$ have been varied, while for the notch depth $a$, a constant value equal to 10 mm has been assumed. Three values of the opening angle $2\alpha$ have been considered: $60^\circ$, $120^\circ$ and $135^\circ$. The notch tip radius assumes for every opening angle three values: 0.5, 1 and 6 mm. The plate has a constant height, $H$, equal to 192 mm and a width, $W$, equal to 100 mm. The numerical results have been obtained thanks to the implementation of the new developed method and its equations in MATLAB®. In the same time, a 2D finite element analysis has been carried out through ANSYS. The Solid 8 node 183 element has been employed and plane stress condition is assumed. The material elastic ($E$, $\nu$, $\sigma_{ys}$) and Norton Creep power law ($n$, $B$) properties are reported in Tab. 1.

For the sake of brevity, only few examples are reported in Fig. 2 (a-b). All the other cases present the same trend of Fig. 2. The theoretical results are in good agreement with the numerical FE values. All the stresses and strains as a function of time have been predicted with acceptable errors. In detail, maximum discrepancy in modulus of 20% has been found for both quantities, with a medium error about 10%. Considering the examples of the strain prediction depicted in Fig. 2(b), the discrepancy within numerical and predicted solution of the strain is most likely due to different approximations introduced in the theoretical formulation, such as the assumption of the elastic-perfectly plastic behavior of the material inside the plastic zone and the employment of the Irwin’s method to estimate the plastic radius.

<table>
<thead>
<tr>
<th>$E$ (MPa)</th>
<th>$\nu$</th>
<th>$\sigma_{ys}$ (MPa)</th>
<th>$n$</th>
<th>$B$ (MPa·h)</th>
</tr>
</thead>
<tbody>
<tr>
<td>191000</td>
<td>0.3</td>
<td>275.8</td>
<td>5</td>
<td>1.8</td>
</tr>
</tbody>
</table>

Table 1: Mechanical properties.

Figure 2: Comparison between theoretical and FEM evolution of stress and strain as a function of time for V-notch geometry: (a) $\rho = 0.5$ mm, $2\alpha = 120^\circ$; (b) $2\alpha = 135^\circ$ and $\rho = 0.5$, 1, 6 mm.
CONCLUSIONS

The present paper proposed an extension of the method presented by Nuñez and Glinka [13] for blunt U-notches, to a blunt V-notches. The key to getting the extension to blunt V-notches is the substitution of the Creager and Paris [15] equations with the more general Lazzarin and Tovo [16] equations that allow an unified approach to the evaluation of linear elastic stress fields in the neighbourhood of crack and notches. The main advantage of the new formulation is that it permits a fast evaluation of the stresses and strains at notches under creep conditions, without the use of complex and time-consuming FE non-linear analyses. It is presented for blunt V-notches but also valid for U-notches. Moreover, the localized creep formulation can be easily derived neglecting the contribution of the far field. The results have shown a good agreement between numerical and theoretical results. Thanks to the extension to blunt V-notches, all geometries can be easily treated with the aim of the numerical method developed.

Although Lazzarin and Tovo equations are valid also in case of sharp V-notches (i.e. for a notch radius that tends to be zero), the values of stress and strain are no longer suitable as characteristic parameters governing failure. As well known, in fact, these local approaches failed when the stress fields tends toward infinity (such as for crack or sharp notches), and the development of alternative solutions becomes crucial. The evaluation of stress and strain at some points ahead of the not tip may be a possible way to address the problem. Different methods are available in literature dealing with this matter, for example based on energy local approaches such as Strain Energy Density [23-25]. This parameter could be useful also to characterize creeping conditions if combined with the present model, giving the possibility to include in the analysis also cracks and sharp V-notches. However, some points remain open:

- order singularity variation with time: when considering creeping conditions, the singularity order does not assume a constant value, but varies with time.
- evolution against time from elastic to elastic-plastic or fully plastic state of the system, especially when dealing with high temperature.

Because of the promising results showed in the preliminary analyses, the authors still devoting effort to overcome the problems cited previously and to combine successfully the proposed model for the prediction of stresses and strain with the SED averaged over a control volume, in order to give a useful and more general tool when dealing with notches subjected to creep, regardless of the specimen geometries.

REFERENCES


NOMENCLATURE

- $a$: notch depth
- $C_p$: plastic zone correction factor
- $d$: distance from the coordinate system origin at which the far field contribution is evaluated
- $E$: Young's modulus
- $K_{Ω}$: strain energy concentration factor
- $K_r$: stress concentration factor
- $K_I$: mode I stress intensity factor
- $r$: radial coordinate
- $r_0$: distance within notch tip and coordinate system origin
- $r_p$: plastic zone radius
- $t$: time
- $2α$: notch opening angle
- $Δε_{c2}^{n}$: creep strain increment at the notch tip at step n
- $Δε_{c2}^{n}$: incremental far field creep strain
- $Δε_{T2}^{n}$: increment of total strain
- $Δr_p$: plastic zone increment
- $Δσ_{c2}^{n}$: stress decrement at the notch tip at step n
- $Δt$: time increment
- $ε^{p0}$: plastic strain at time $t = 0$
- $ε_{c2}^{i}$: creep strain at the far field
- $ε_{c2}^{t}$: creep strain at the notch tip
- $ε_{c2}^{t}$: time dependent notch tip strain

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\( \varepsilon^{o} \) actual elastic-plastic strain
\( \varepsilon^{'} \) hypothetical strain components obtained from linear elastic analysis
\( \theta \) angular coordinate
\( \lambda_{1} \) mode I eigenvalue
\( \mu_{1} \) mode I second order eigenvalue
\( \rho \) notch tip radius
\( \sigma_{max} \) maximum stress at the notch tip
\( \sigma_{nom} \) applied nominal stress
\( \sigma^{'}_{22} \) far field stress
\( \sigma^{'}_{22} \) far field stress, \( t = 0 \)
\( \sigma^{''}_{22} \) time dependent notch tip stress
\( \sigma^{a}_{q} \) actual elastic-plastic stress
\( \sigma^{q}_{\varepsilon} \) hypothetical stress components obtained from the linear elastic analysis
\( \chi_{1} \) mode I associated constant